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When forces are continuously distributed over a region of a structure, the cumulative effect of this distribution must be determined. The designers of high-performance sailboats consider both air-pressure distributions on the sails and water-pressure distributions on the hull.



DISTRIBUTED Forces

CHAPTER OUTLINE

5/1 Introduction

5

SECTION A CENTERS OF MASS AND CENTROIDS

- 5/2 Center of Mass
- 5/3 Centroids of Lines, Areas, and Volumes
- 5/4 Composite Bodies and Figures; Approximations
- 5/5 Theorems of Pappus

SECTION B SPECIAL TOPICS

- 5/6 Beams-External Effects
- 5/7 Beams–Internal Effects
- 5/8 Flexible Cables
- 5/9 Fluid Statics
- 5/10 Chapter Review

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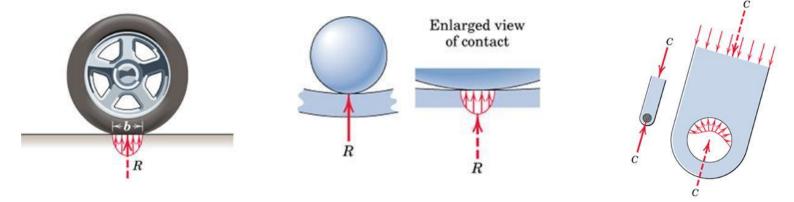
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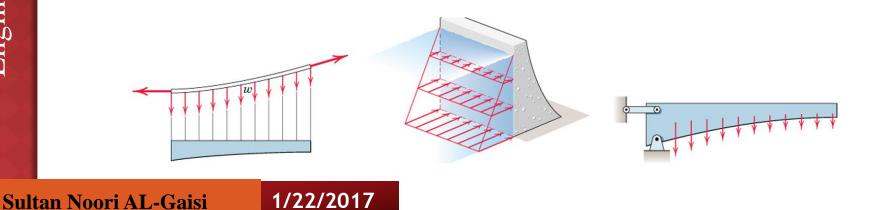
5/1 INTRODUCTION

- In the previous chapters we treated all forces as concentrated along their lines of action and at their points of application.
- This treatment provided a reasonable model for those forces. Actually,
 "concentrated" forces do not exist in the exact sense, since every external
 force applied mechanically to a body is distributed over a finite contact
 area, however small.

Concentrated Forces: If dimension of the contact area is negligible compared to other dimensions of the body the contact forces may be treated as Concentrated Forces



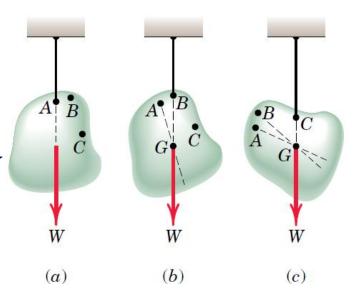
Distributed Forces: If forces are applied over a region whose dimension is not negligible compared with other pertinent dimensions proper distribution of contact forces must be accounted for to know intensity of force at any location.



SECTION A CENTERS OF MASS AND CENTROIDS

5/2 CENTER OF MASS

- If we suspend the body, as shown in Fig. 5/3, from any point such as *A*, the body will be in equilibrium under the action of the tension in the cord and the resultant *W* of the gravitational forces acting on all particles of the body.
- This resultant is clearly collinear with the cord.
- Dotted lines show lines of action of the resultant force in each case.
- These lines of action will be concurrent at a single point *G*, which is called the *center of gravity* of the body.



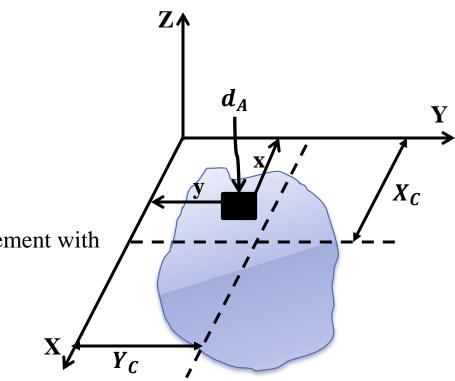
Centroid of an Area

The point located by the coordinates X_c and Y_c is defined as the centroid of the area. The centroid must lie in the plane of the area and two coordinates are sufficient to locate it by the following procedure

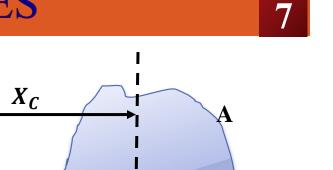
- 1. Take an element wit area (dA).
- 2. Find the total area of the body (A).

 $A = \int d_A \dots \dots (5.1)$

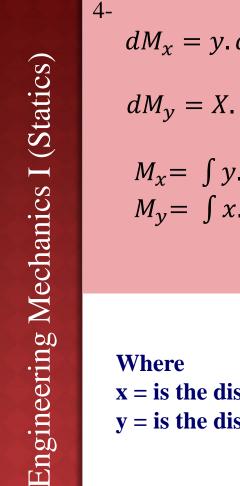
3. Determine the first moment of the element with respect X-axis and Y-axis.



6



Χ



$dM_{x} = y \cdot d_{A}$ $dM_{y} = X \cdot d_{A}$ $M_{x} = \int y \cdot d_{A}$ $M_{y} = \int x \cdot d_{A}$

x = is the distance from the centroid of the element to the y-axis.

 $\mathbf{y} = \mathbf{is}$ the distance from the centroid of the element to the y-axis

5. Find the coordinates of the centroid of the body as follows:-

$$X_{c} = \frac{\int x \, dA}{\int dA} = \frac{M_{y}}{A}$$
$$y_{c} = \frac{\int y \, dA}{\int dA} = \frac{M_{x}}{A}$$

The first moment of an area about on axis has units of mm^3, cm^3, m^3. the area is s scalar quantity and the sense of moment of an area depends on the moment arm, so, its positive if the area is on one side of the opposite side if the axis (see fig 5-2 b, c)

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Engineering Mechanics I (Statics)

Ex:- find the centroid of rectangle has a, b dimensions, as shown in figure below.

Solution b To find ye, take element with depth (dy) dy? 6 yc= $\int \frac{y}{dA} = \frac{Mx}{A}$ 0 $A = \int dA = \int q \cdot dy = q \cdot y \int$ A= ab-9+0 $\int y(ady) = \frac{ay^2}{2} \Rightarrow Mx = \frac{ab^2}{2}$ 5 y - dA = $\frac{Mx}{A} = \frac{ab^2/2}{ab^2} = \frac{b}{2}$

(2) To find Xc, take element with respect width (dx)

$$X_{c} = \frac{\int X \cdot dA}{\int A} = \frac{M \cdot y}{A}$$

$$M_{y} = \int_{0}^{q} X \cdot dA = \int_{0}^{q} X (b \cdot dx) = \frac{bx^{2}}{2} \int_{0}^{q} \frac{dx}{dx}$$

$$M_{y} = \frac{a^{2}b/2}{A} = \frac{a^{2}b/2}{a \cdot b} = \frac{a}{2}$$

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Engineering Mechanics I (Statics)

y

For 3D

$$X_{c} = \frac{\int x \, dA}{\int dA} = \frac{M_{yz}}{A}$$
$$y_{c} = \frac{\int y \, dA}{\int dA} = \frac{M_{xz}}{A}$$

$$z_c = \frac{\int z \, dA}{\int dA} = \frac{M_{xy}}{A}$$

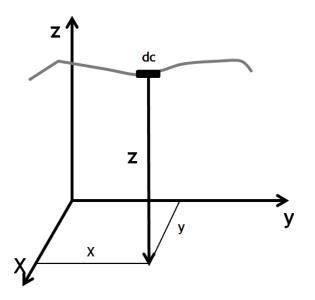
Centroid of curve surface centroid of the

$$X_{c} = \frac{\int x \, dL}{\int dL}$$
$$y_{c} = \frac{\int y \, dL}{\int dL}$$
$$z_{c} = \frac{\int z \, dL}{\int dL}$$
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 d_A Ζ Х

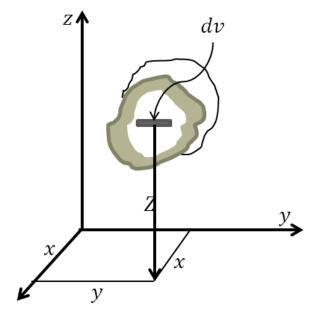
Z▲



Engineering Mechanics I (Statics)

$$X_c = \frac{\int x \, dv}{\int dv} \qquad y_c = \frac{\int y \, dv}{\int dv}$$

$$z_c = \frac{\int z \, dv}{\int dv}$$



$$dm = \rho . dv$$

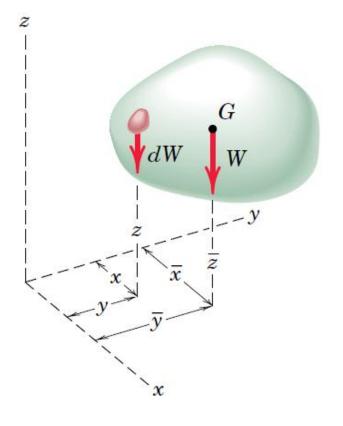
$$X_{m} = \frac{\int x \, dm}{\int dm} = \frac{\int x \, \rho. \, dv}{\int \rho. \, dv}$$
$$y_{m} = \frac{\int y \, dm}{\int dm} = \frac{\int y \, \rho. \, dv}{\int \rho. \, dv}$$
$$z_{m} = \frac{\int z \, dm}{\int dm} = \frac{\int z \, \rho. \, dv}{\int \rho. \, dv}$$

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Center of gravity

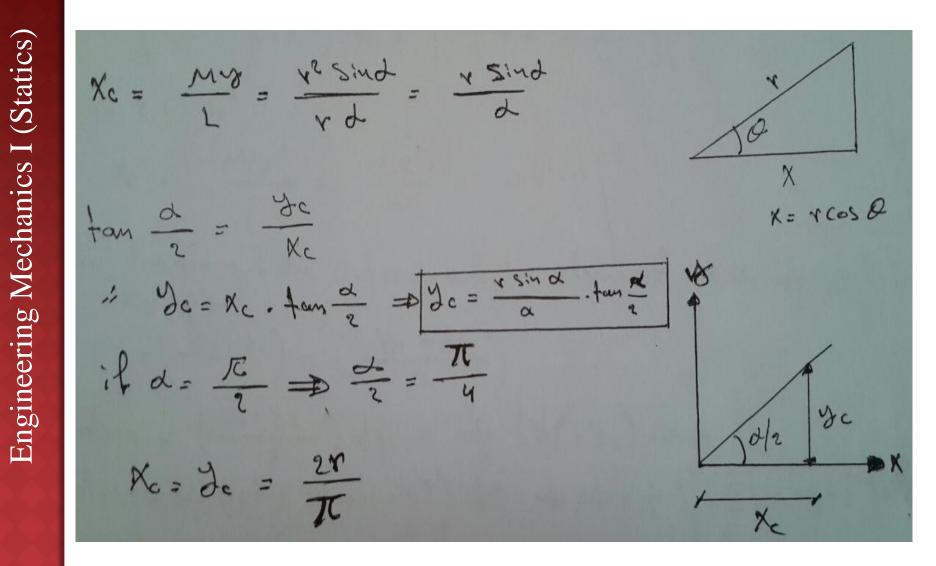
$$dw = gdm = \rho. g. dv$$

$$X_{w} = \frac{\int x \, dw}{\int dw} = \frac{\int x \, \rho. \, g. \, dv}{\int \rho. \, dv}$$
$$y_{w} = \frac{\int y \, dw}{\int dw} = \frac{\int y \, \rho. \, g. \, dv}{\int \rho. \, dv}$$
$$z_{w} = \frac{\int z \, dw}{\int dw} = \frac{\int z \, \rho. \, g. \, dv}{\int \rho. \, dv}$$



Ex:- determine the coordinate of the centroid of the circular arc(line), in figure below.

* from Symmetry, the Controid must lie on about through the origion at an angle d/2 with X-adis. P * The length of the element is 10 dL=rdQ - L = SdL = Srdd = rdl ~ L= ra dMy = x. dL = (r cos Q)(rdQ) = r2 cos Q dQ $My = \int r^2 \cos \theta \, d\theta = r^2 \sin \theta \implies My = r^2 \sin \theta$

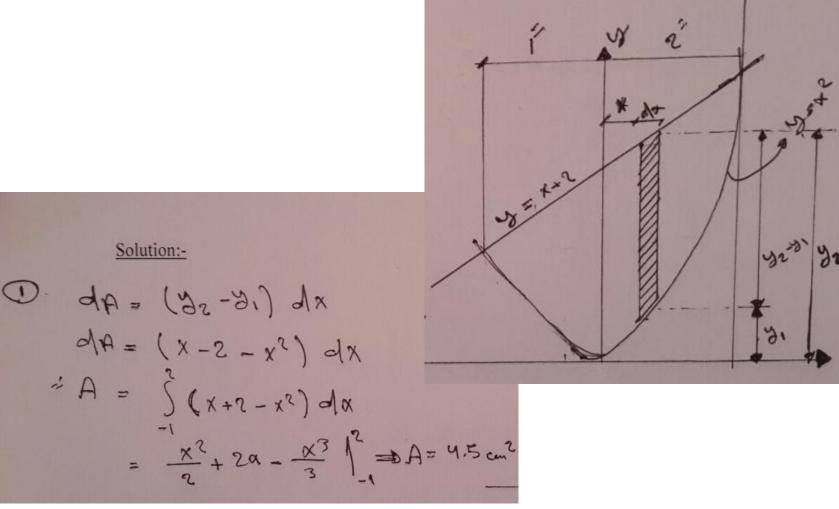


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Ex:- Determine the coordinate of the centroid of the area in figure below, bounded by the curve $y = x^2$ and the line y = x + 2. in the equations x and y are in inches.



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(2) The moment of element with respect to fig

$$y_{-\alpha x is}$$

 $dim y = \chi \cdot d\mu = \chi (\chi + \chi - \chi^2) dx = (\chi^2 + \chi - \chi^3) dx$
 $My = \int_{-1}^{2} (\chi^2 + \chi - \chi^3) dx = \left[\frac{\chi^3}{3} + \chi^2 - \frac{\chi^4}{4}\right]_{1-1}^{2}$
 $My = 2.25 \text{ in}^3$
 $\therefore \chi_{c} = \frac{My}{4} = \frac{2.25}{4.5} = 0.5 \text{ in}$

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Engineering Mechanics I (Statics)

The location at the activity of the elements with respect to

$$xy \cdot pbn$$
.
 $(3 + \frac{3}{2} - 3) = (\frac{3}{2} + 32)$
 $dMx = (\frac{3}{2} + 32) d\mu = \frac{1}{2} (3 + 32) \cdot d\mu$
 $dMx = \frac{1}{2} (x^2 + 4x + 4 - x^4) dx = Mx = \frac{1}{2} \int_{-1}^{2} (x^2 + 4x + 4 - x^2) dx$
 $Mx = \frac{1}{2} [\frac{x^3}{3} + 2x^2 + 4x - \frac{x^5}{5}]_{-1}^{2} = Mx = 7 \cdot 20 \text{ in}^{3}$
Sultan Noori $y_c = \frac{7 \cdot 20 \text{ in}^{3}}{4} = 1.6 \text{ in}$ Page 9